

POWER SYSTEM CONTROL

INTRODUCTION:

The preceding chapters were devoted to problems associated with the selection of a normal operating state for the power system and optimum scheduling of generation. The present chapter deals with the continuous control of active and reactive power in order to keep the system in steady state. The power system being dynamic, the demand continuously deviates from its normal value. This leads to a small change in the state of the system. The automatic control should act in a closed loop manner, to detect these changes and initiate actions to eliminate the deviations. Briefly stated, the control strategy should be designed to deliver power to an interconnected system economically and reliably, while maintaining the voltage and frequency within the permissible limits.

Changes in real power mainly affect the system frequency and changes in reactive power mainly depend on changes in voltage magnitude and are relatively less sensitive to changes in frequency. Thus, real and reactive powers can be controlled separately. The Automatic Load Frequency Control (ALFC) controls the real power and the Automatic Voltage Regulator (AVR) regulates the voltage magnitude and hence the reactive power. The two controls, along with the generator and prime mover are shown in Fig.1. Unlike the AVR, ALFC is not a single loop. A fast primary loop responds to the frequency changes and regulates the steam (water) flow via the speed governor and control valves to match the active power output with that of the load. The time period here is a few seconds. The frequency is controlled via control of the active power.

A slower secondary loop maintains fine frequency adjustment to maintain proper active power exchange with other interconnected networks via tie-lines. This loop does not respond to fast load changes but instead focuses on changes, which lead to frequency drifting over several minutes.

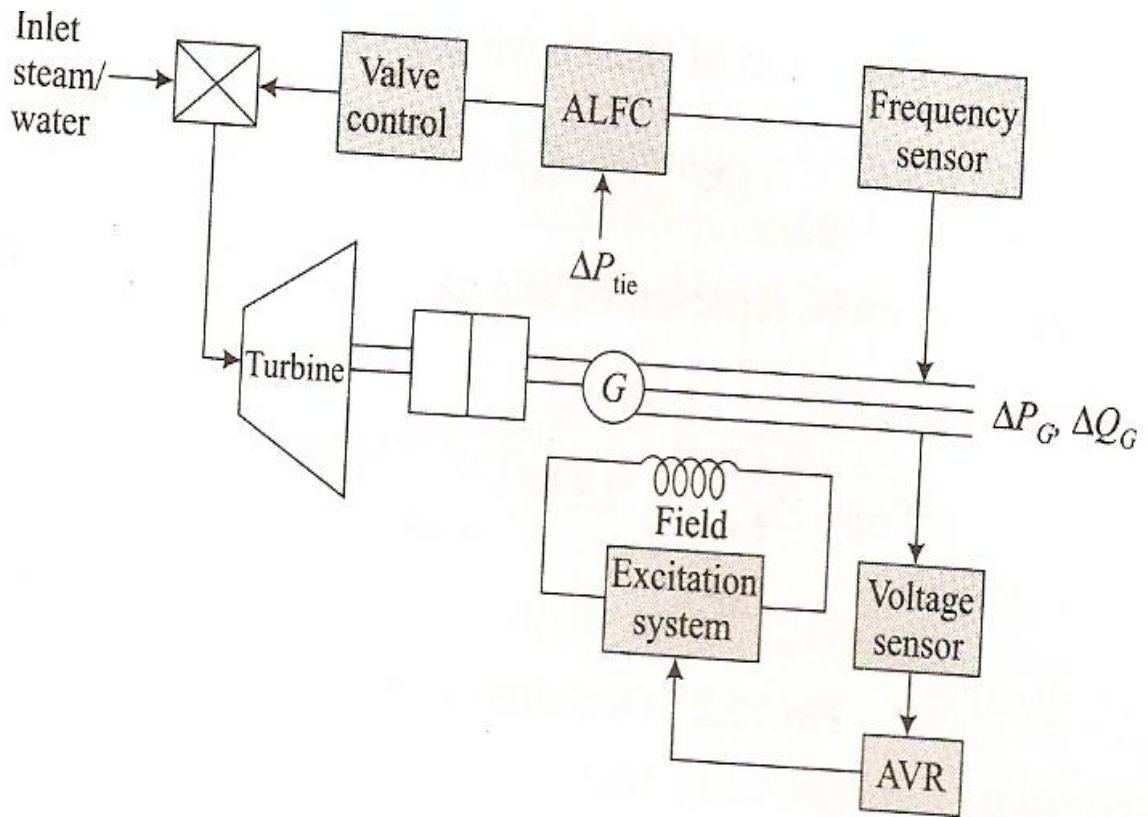


Fig.1 ALFC and AVR

Since the AVR loop is much faster than the ALFC loop, the AVR dynamics settle down before they affect the ALFC control loop. Hence, cross-coupling between the controls can be neglected. With the growth of large interconnected systems, ALFC has gained importance in recent times. This chapter presents an introduction to power system controls.

AUTOMATIC LOAD FREQUENCY CONTROL:

The functions of the ALFC are to maintain steady frequency, control tie-line power exchange and divide the load between the generators. The tie-line power deviation is given by P_{tie} and the change in frequency f , is measured by Δf , the change in the rotor angle δ . The error signals Δf and P_{tie} are amplified, mixed and transformed to a

real power signal, which controls the valve position to generate a command signal P_v .

P_v is sent to the prime mover to initiate change in its torque. The prime mover changes the generator output by P_G , so as to bring f and P_{tie} within acceptable limits. The next step in the analysis is to build the mathematical model for the ALFC.

Generator Model:

We can apply the swing equation to a small perturbation to obtain the linearized equation.

$$\frac{2H}{\tilde{S}_s} \frac{d^2 \Delta u}{dt^2} = P_m - P_e$$

Expressing speed deviation in pu, can be written as

$$\frac{d\Delta\tilde{S}}{dt} = \frac{1}{2H} (P_m - P_e)$$

Taking the Laplace Transform of we get

$$\Delta\tilde{S}(s) = \frac{1}{2Hs} (P_m(s) - P_e(s))$$

Load Model:

The details of load modeling are covered in chapter 11. In general, the loads are composite. Resistive loads such as lighting and heating loads are independent of frequency. However, in case of electric motors, the power is dependent on frequency. We can arrive at a composite frequency dependent load characteristic given by

$$P_e = P_L + D$$

where P_L = non frequency sensitive load change

D = Load damping constant

D = frequency sensitive load change

The damping constant is expressed as percent change in load for one percent change in frequency. A value of $D = 1.2$ means a change in frequency by 1% causes the load to change by 1.2%.

Turbine Model:

The prime mover for the generator is the turbine, which is mostly a steam turbine or a hydro turbine. In the simplified model, the turbine can be represented by a first order, single time constant transfer function given by

$$G_T(s) = \frac{\Delta P_m(s)}{\Delta P_v(s)} = \frac{1}{1 + \tau_T s}$$

where P_v = Change in valve output

τ_T = turbine time constant

τ_T varies from 0.2 – 2 secs. The exact value of τ_T , depends on the type of turbine.

Governor Model:

If the electrical load on the generator suddenly increases, the output electrical power exceeds the input mechanical power. The difference is supplied by the kinetic energy stored in the system. The reduction in the kinetic energy causes the turbine speed and frequency to fall. The turbine governor reacts to this change in speed, and adjusts the turbine input valve/gate to change the mechanical power output to match the increased power demand and bring the frequency to its steady state value. Such a governor which brings back the frequency to its nominal value is called as *isochronous governor*. The essential elements of a conventional governor system are shown in Fig

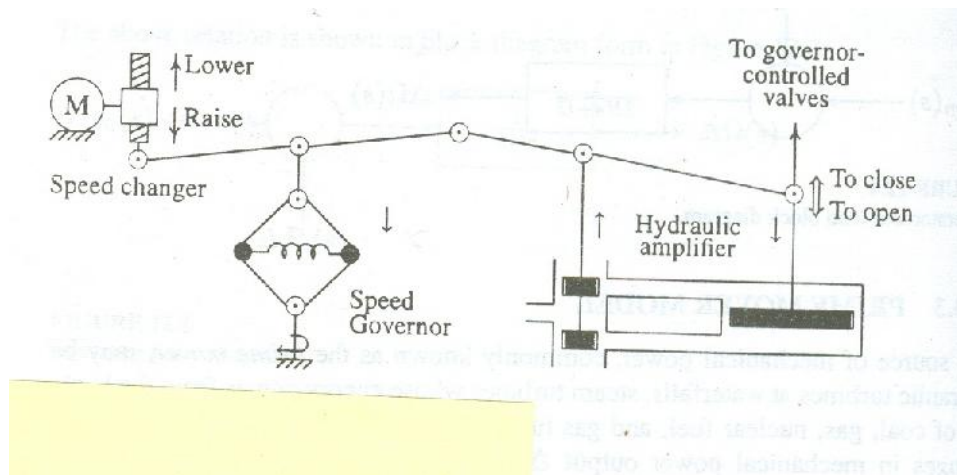


Fig Conventional governor

The major parts are

- (i) **Speed Governor:** This consists of centrifugal flyballs driven directly or through gears by the turbine shaft, to provide upward and downward vertical movements proportional to the change in speed.
- (ii) **Linkage mechanism:** This transforms the flyball movement to the turbine valve through a hydraulic amplifier and provides a feed back from turbine valve movement.
- (iii) **Hydraulic amplifiers:** These transform the governor movements into high power forces via several stages of hydraulic amplifiers to build mechanical forces large enough to operate the steam valves or water gates.
- (iv) **Speed changer:** This consists of a servomotor which is used to schedule the load at nominal frequency. By adjusting its set point, a desired load dispatch can be scheduled.

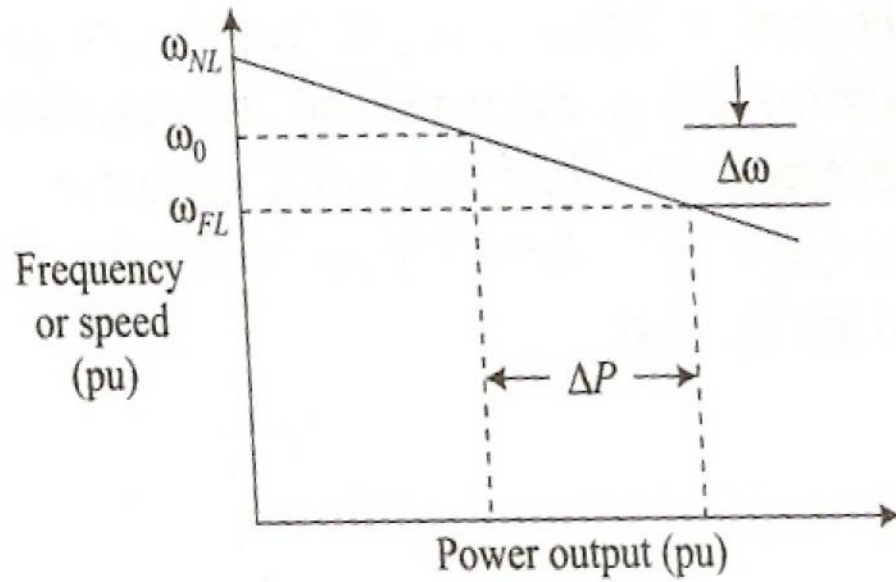
An isochronous governor works satisfactorily only when a generator is supplying an isolated load, or when only one generator is required to respond to change in load in a multi generator system. For proper power sharing between a number of generators connected to the system, the governors are designed to permit the speed to drop as the load is increased. This provides the speed – output characteristic a droop as shown in Fig. The speed regulation R is given by the slope of the speed – output characteristic.

$$R = \frac{\Delta \check{S}}{\Delta P}$$

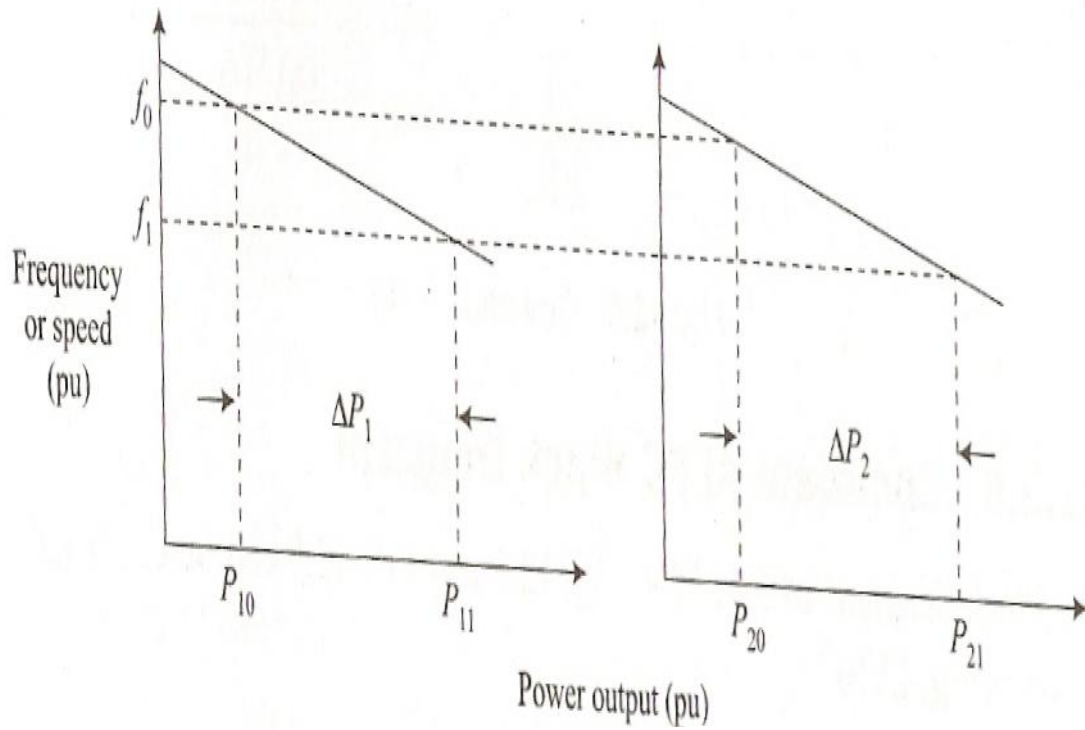
Governor % speed regulation is defined as

$$\%R = \left(\frac{\check{S}_{NL} - \check{S}_{FL}}{\check{S}_o} \right) \times 100$$

where \check{S}_{NL} = No-load speed
 \check{S}_{FL} = Full load speed
 \check{S}_o = Nominal speed



To illustrate how load is shared between two generators, consider two generators with droop characteristics as shown in Fig below



Let the initial frequency be f_0 and the outputs of the two generators be P_{10} and P_{20} respectively. If now the load increases by an amount P_L , the units slow down and the governors increase the output until a common operating frequency f_1 is reached. The amount of load picked up by each generator to meet the increased demand P_L depends on the value of the regulation.

$$P_1 = \frac{\Delta f}{R_1}$$

$$P_2 = \frac{\Delta f}{R_2}$$

$$\frac{\Delta P_1}{\Delta P_2} = \frac{R_2}{R_1}$$

The output is shared in the inverse ratio of their speed regulation. The output of the speed governor is P_g , which is the difference between the set power P_{ref} and the power $\frac{\Delta \dot{S}}{R}$ which is given by the governor speed characteristic.

$$P_g = P_{ref} - \frac{\Delta\check{S}}{R}$$

$$P_g(s) = P_{ref}(s) - \frac{\Delta\check{S}(s)}{R}$$

The hydraulic amplifier transforms the command into valve/gate position P_v . Assuming a time constant τ_g for the governor,

$$P_v(s) = \frac{1}{1 + \tau_g s} \Delta P_g(s)$$

Complete ALFC block diagram:

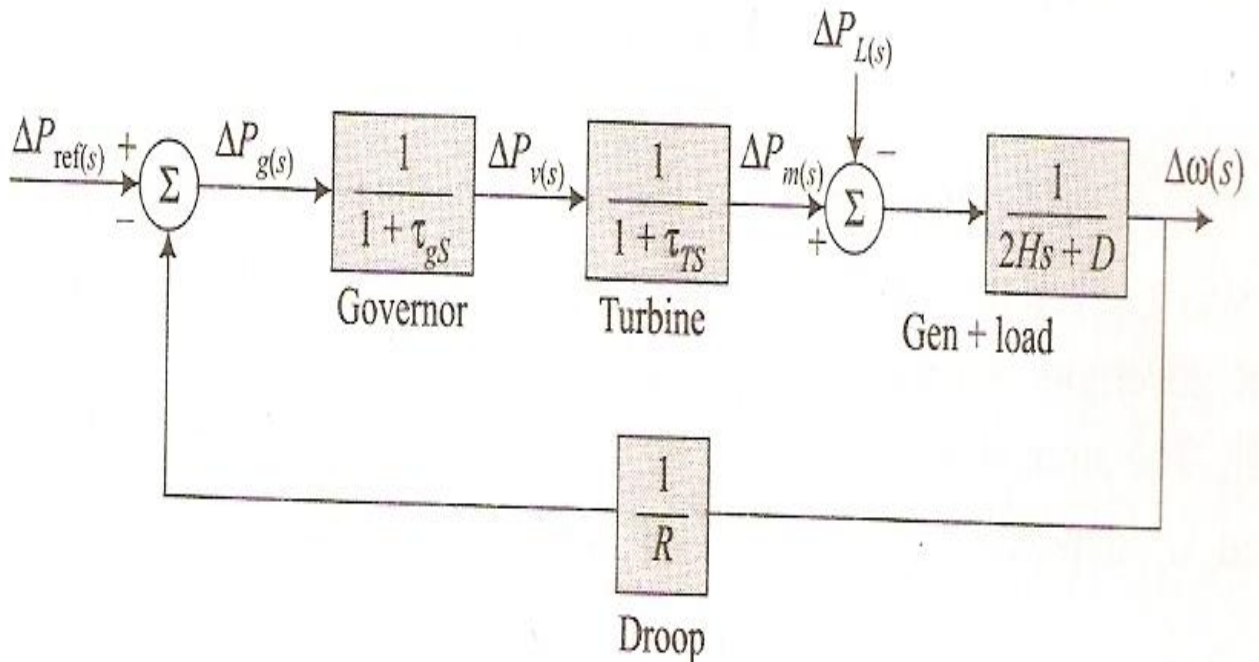
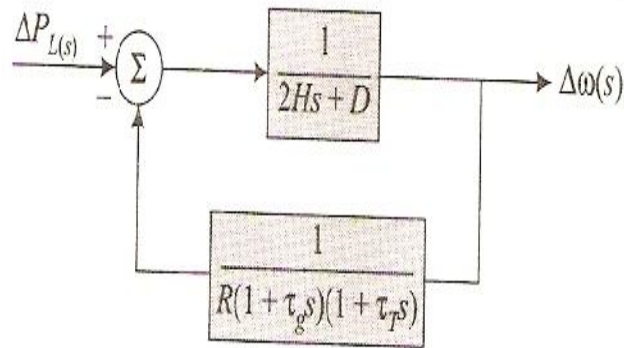


Fig Block diagram of complete governor system

The Figure shows the complete load frequency control for an isolated generator supplying a load. Since we are interested in the change in speed for change in load, we can obtain the transfer function $\frac{\Delta\check{S}(s)}{-\Delta P_L(s)}$ from Fig below which is a reduced order model.



Reduced block diagram

The closed loop transfer function is obtained from Fig

$$\begin{aligned} \frac{\Delta\omega(s)}{-\Delta P_L(s)} &= \frac{\left(\frac{1}{2Hs + D}\right)}{1 + \left(\frac{1}{2Hs + D}\right)\left(\frac{1}{R(1 + \tau_g s)(1 + \tau_T s)}\right)} \\ &= \frac{(1 + \tau_g s)(1 + \tau_T s)}{(2Hs + D)(1 + \tau_g s)(1 + \tau_T s) + \frac{1}{R}} \end{aligned}$$

We can write

$$\Delta\omega(s) = \Delta P_L(s) T(s)$$

If we consider a step change P_L in the load,

$$P_L(s) = \frac{\Delta P_L}{s}$$

The steady state frequency deviation ω_{ss} is given by the limit of $\Delta\omega(s)$ as $t \rightarrow \infty$. This can be obtained by application of the final value theorem

$$\begin{aligned} \omega_{ss} &= \lim_{t \rightarrow \infty} \Delta\omega(s) = \lim_{s \rightarrow 0} s \Delta\omega(s) \\ &= \lim_{s \rightarrow 0} s \left(\frac{-\Delta P_L}{s} T(s) \right) \end{aligned}$$

Now
$$\lim_{s \rightarrow 0} T(s) = \frac{1}{D + \frac{1}{R}}$$

$$ss = (\Delta P_L) \frac{1}{D + \frac{1}{R}}$$

If there are no frequency sensitive loads, $D = 0$; in which case

$$ss = \frac{-\Delta P_L}{\frac{1}{R}}$$

The steady state speed deviation thus, depends on the governor speed regulation. If several generators are connected to the system, the composite frequency – power characteristics depends on the combined effect of the droops of all the generator speed governors. If we consider n generators with a composite load damping coefficient D , the steady state speed deviation after a load change ΔP_L is given by

$$ss = \frac{-\Delta P_L}{D + \left(\frac{1}{R_1} + \frac{1}{R_2} + \dots + \frac{1}{R_n} \right)} = \frac{-\Delta P_L}{D + \frac{1}{R_{eq}}}$$

where
$$\frac{1}{R_{eq}} = \frac{1}{R_1} + \frac{1}{R_2} + \dots + \frac{1}{R_n}$$

The *stiffness* of the system, is given by

$$= \frac{-\Delta P_L}{\Delta \check{S}_{ss}} = \frac{1}{R_{eq}} + D \text{ MW/Hz}$$

is also called the *frequency bias factor* and is indicative of the change in frequency which would occur for a change in the load.

(* The pu speed deviation is same as pu frequency deviation f)

Example 1: A system consists of 4 identical 250 MVA generators feeding a load of 510 MW. The inertia constant H of each unit is 2.5 on the machine base. The total load varies by 1.4% for a 1% change in frequency. If there is a drop in load of 10MW, determine the system block diagram expressing H and D on a base of 1000MVA. Give expression for the speed deviation, assuming there is no speed governor.

Solution:

$$H \text{ for 4 units on 1000MVA base} = 4 \times 2.5 \times \frac{250}{1000} = 2.5$$

$$\text{Load after drop of 10 MW} = 510 - 10 = 500 \text{ MW}$$

D for load on base of 1000 MVA is given by

$$D = 1.4 \times \frac{500}{1000} = 0.7\%$$

[note that a change of load of 1.4% on base 500 MW corresponds to 0.7% on base of 1000MVA]

The standard first order transfer function form is given by $\frac{K}{1 + sT}$. In the reduced order model, the feedback loop is zero, since no governor is modeled. Substituting the values, and expressing in standard form we get

The gain = 1.428 and time constant = 7.14 secs.

$$P_L = -10 \text{ MW} = \frac{-10}{1000} = -0.01 \text{ pu.}$$

$$P_L(s) = \frac{-0.01}{s} \text{ pu}$$

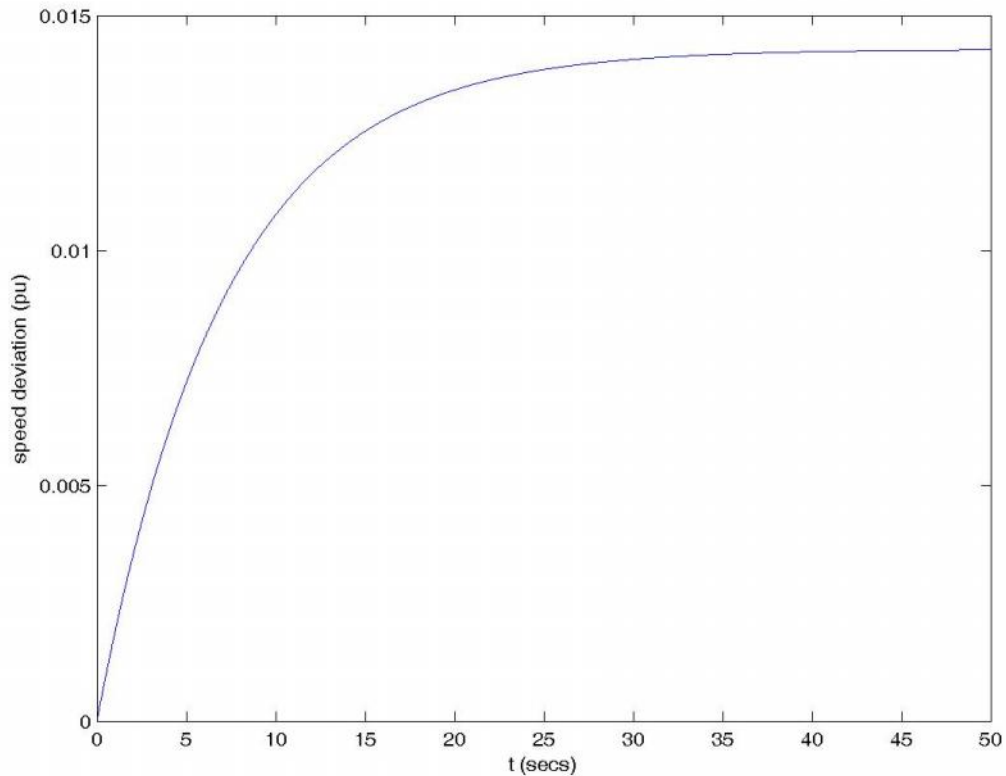
From block diagram

$$\begin{aligned} (s) &= -\left(\frac{-0.01}{s}\right)\left(\frac{1.428}{1+7.14s}\right) \\ &= \left(\frac{0.01}{s}\right)\left(\frac{1.428}{1+7.14s}\right) = \frac{0.01428}{s} - \frac{0.10196}{1+7.14s} \\ &= \frac{0.01428}{s} - \frac{0.01428}{s + 0.14} \end{aligned}$$

Taking inverse laplace transform

$$(t) = 0.01428 (1 - e^{-0.14t})$$

The pu speed deviation as function of time is shown in Fig below.



The steady state speed deviation is 0.01428 pu. If frequency is 50 Hz, steady state frequency deviation = $50 \times 0.01428 = 0.714$ Hz. The frequency deviation is positive since a decrease in load leads to increase frequency.

Example 2.

An isolated generator and its control have following parameters

Generator inertia constant = 5sec

Governor time constant $t_g = 0.25$ sec

Turbine time constant $t_T = 0.6$ sec

Governor speed regulation = 0.05 pu

$D = 0.8$

The turbine rated output is 200 MW at 50 Hz. The load suddenly increases by 50 MW. Find the steady state frequency deviation. Plot the frequency deviation as a function of time.

Solution:

The transfer function is given by

$$\frac{\Delta\tilde{S}(s)}{-\Delta P_L(s)} = T(s) = \frac{(1 + 0.25s)(1 + 0.6s)}{(10s + 0.8)(1 + 0.25s)(1 + 0.6s) + \frac{1}{0.05}}$$

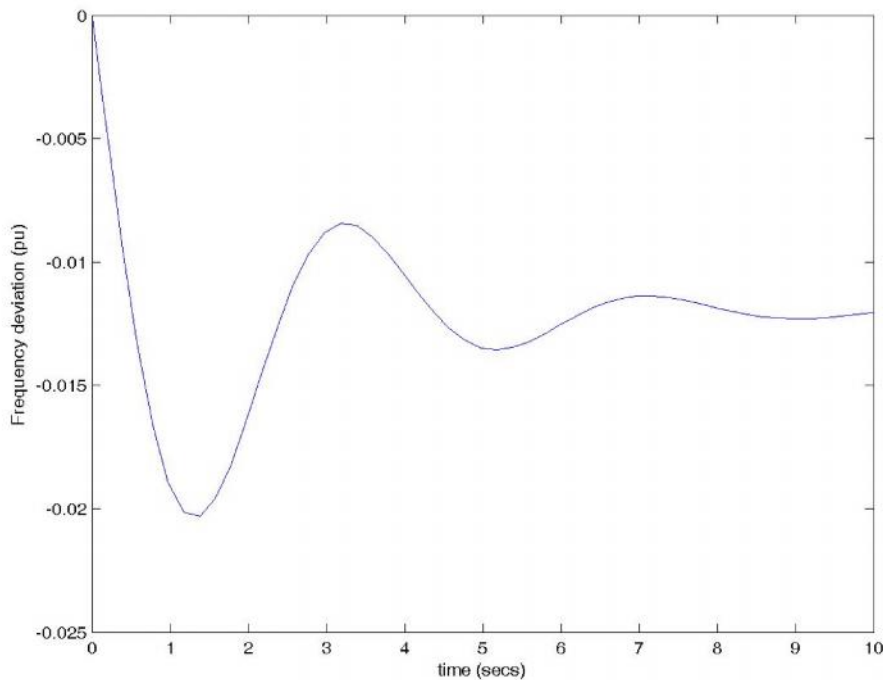
$$= \frac{0.15s^2 + 0.85s + 1}{1.5s^3 + 8.62s^2 + 10.68s + 20.8}$$

$$P_L = \frac{50}{200} = 0.25 \text{ pu}$$

$$s_{ss} = \frac{-\Delta P_L}{D + \frac{1}{R}} = \frac{-0.25}{0.8 + \frac{1}{0.05}} = 0.01202 \text{ pu.}$$

Steady state frequency deviation $f_{ss} = 0.01202 \times 50 = 0.601 \text{ Hz.}$

The frequency decreases since the load has increased. The time response to a step deviation of 0.25 is shown Fig



Frequency deviation for step response for example 2

CONCEPT OF AUTOMATIC GENERATION CONTROL (AGC):

With the primary ALFC, it was seen that a change in the system load results in a steady state frequency deviation, depending on the regulation and frequency sensitivity (as indicated by D) of the load. All the connected generator units of the system contribute to the overall change in generation, irrespective of the location of the load change. Thus, restoration to nominal system frequency requires additional control action which changes the load reference set point to match the variations in the system load. This control scheme is called the Automatic Generation Control (AGC). The main objectives of the AGC are to regulate the system frequency and maintain the scheduled power interchanges, between the interconnected areas, via the tie-lines. A secondary objective of the AGC is to distribute the required change in generation among the various units to obtain least operating costs. During large transient disturbances and emergencies, AGC is bypassed and other emergency controls act.

AGC in a single area:

In a single area system since there is no tie-line schedule to be maintained, the function of AGC is only to bring the frequency to the nominal value. This is achieved by introducing an integral controller to change the load reference setting so as to change the speed set point. The integral controller forces the steady state speed deviation to zero.

The gain K_I of the integral controller needs to be adjusted for satisfactory response in terms of over shoot, setting time etc.

The closed loop transfer function with integral controller is given by

$$T(s) = \frac{\Delta\check{S}(s)}{-\Delta P_L(s)} = \frac{s(1 + \check{t}_g s)(1 + \check{t}_T s)}{s(2Hs + D)(1 + \check{t}_g s)(1 + \check{t}_T s) + K_I + \frac{s}{R}}$$

Example 4:

In example 2, an integral controller with gain $K_I = 6$ is added. Obtain the dynamic response, if all other conditions are same.

Solution:

From the transfer function

$$\frac{\Delta\check{S}(s)}{-\Delta P_L(s)} = \frac{0.15s^3 + 0.85s^2 + s}{1.5s^4 + 8.62s^3 + 10.68s^2 + 20.8s + 6}$$

The change in P_L , $P_L = 0.25$ pu.

The response of the above function for a step change of 0.25 pu is plotted using Matlab. The response is shown if Fig below.

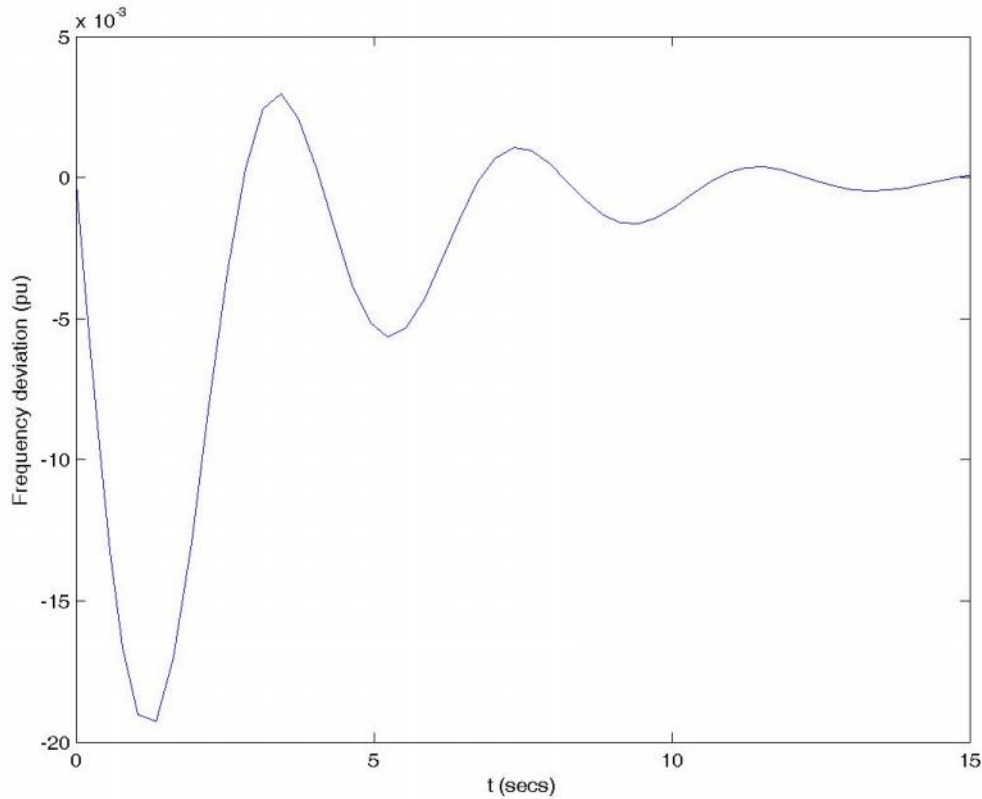


Fig: Dynamic response of example 4

It can be seen that the steady state frequency deviation is now zero. However, the overshoot and setting time are more.

AGC in multi area systems:

In inter connected systems, a group of generators are closely coupled internally and swing in phase. Such a group is called coherent group. The ALFC loop can be represented for the whole area, referred to as control area.

Consider two areas interconnected by lossless tie-line of reactance X_{tie} , with a power flow P_{12} from area 1 to area 2 as shown in Fig. a. Let the generators be represented

by a single equivalent generator for area 1 and area 2. The generators are modeled simply as constant voltage sources behind reactance as shown in Fig. b. We first consider only primary ALFC loop as shown in Fig. c.

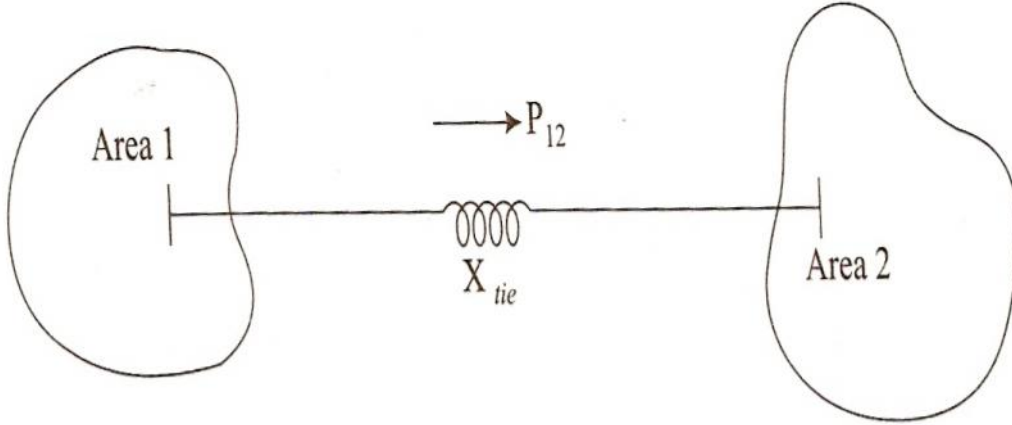


Fig a : Two area system

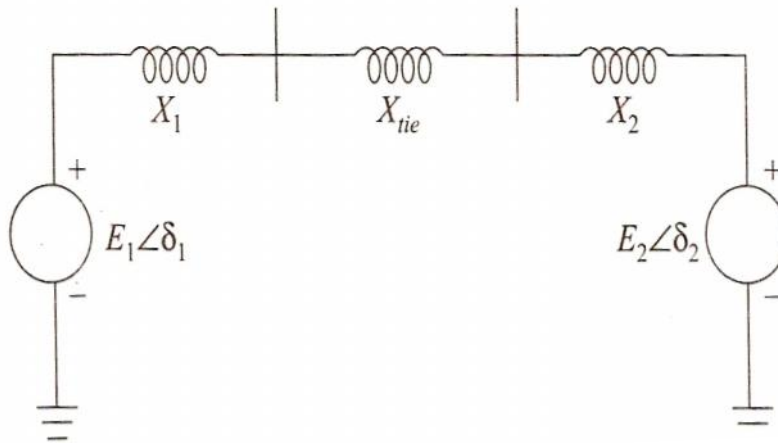


Fig 12.19b: Electrical equivalent

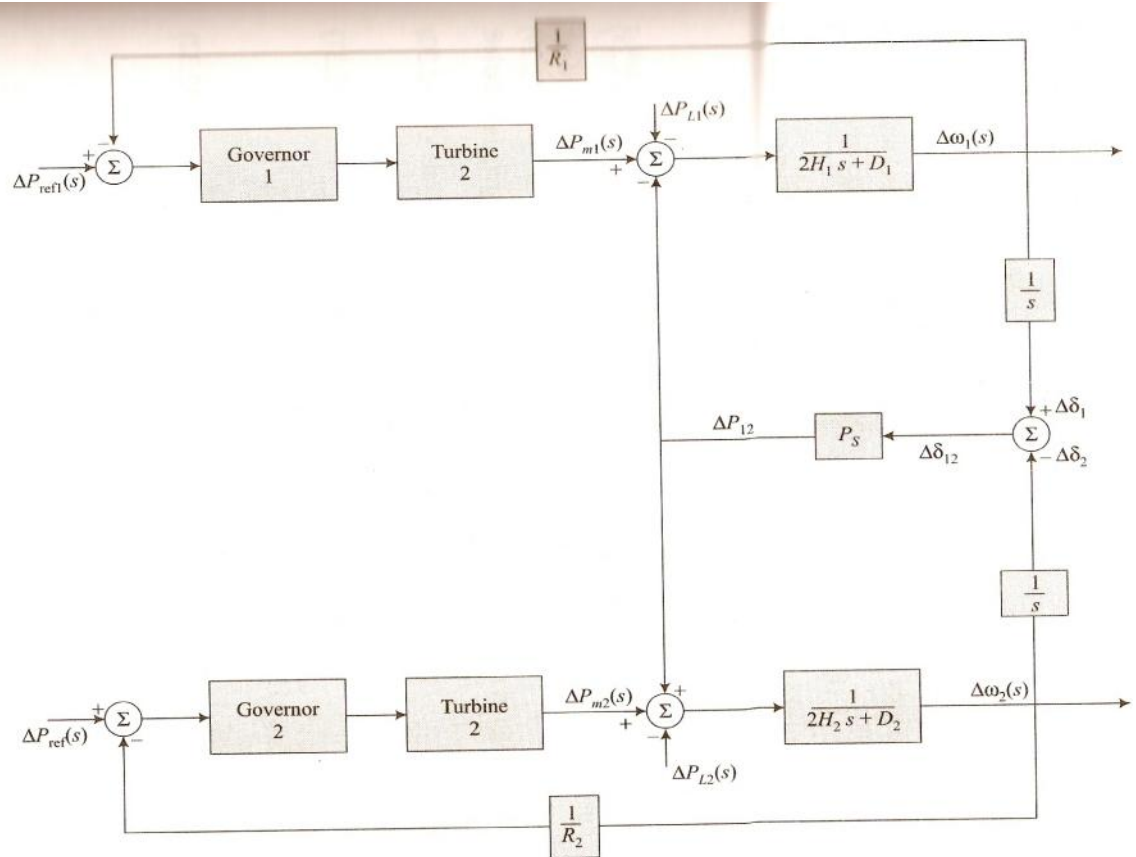


Fig c: Two area system with primary ALFC

H is the equivalent inertia constant of each area. The turbines are represented by the effective speed droop R and load damping constant D.

Under steady state the power transferred over the tie-line is given by

$$P_{12} = \frac{|E_1| |E_2| \sin u_{12}}{X_{12}}$$

where $X_{12} = X_1 + X_{tie} + X_2$ and $u_{12} = \delta_1 - \delta_2$. For a small deviation ΔP_{12} of the tie line power flow, we can write

$$P_{12} = \left. \frac{\partial P_{12}}{\partial u_{12}} \right|_{u_{120}} \Delta u_{12} = P_s \quad \Delta u_{12}$$

where $\left. \frac{\partial P_{12}}{\partial u_{12}} \right|_{u_{120}} = P_S$, is the slope of the power angle curve evaluated at the initial operating point ($u_{120} = u_{10} = u_{20}$) and is the synchronizing power coefficient..

$$P_S = \left. \frac{\partial P_{12}}{\partial u_{12}} \right|_{u_{120}} = \frac{|E_1| |E_2|}{X_{12}} \cos u_{120}$$

$$P_{12} = P_S (u_{12} - u_{120})$$

A positive P_{12} occurs when $u_{12} > u_{120}$ and indicates a flow of real power from area 1 to area 2. This has the effect of increasing load on area 1 and decreasing load on area 2. Hence P_{12} has negative sign for area 1 and positive sign for area 2 in Fig c.

Consider a change in load P_{L1} in area 1. The steady state frequency deviation Δf is same for both the areas. Hence

$$\Delta f = \Delta f_1 = \Delta f_2.$$

For area 1,

$$P_{m1} - P_{12} - P_{L1} = \Delta f D_1$$

For area 2,

$$P_{m2} + P_{12} = \Delta f D_2$$

The change in mechanical powers depends on the respective regulations.

$$P_{m1} = \frac{-\Delta f}{R_1}$$

$$P_{m2} = \frac{-\Delta f}{R_2}$$

Substituting we get

$$\Delta f \left(\frac{1}{R_1} + D_1 \right) = -P_{12} - P_{L1}$$

$\frac{1}{R_1} + D_1 = B_1$, the frequency bias factor for area 1.

$$\Delta f B_1 = -P_{12} - P_{L1}$$

Similarly

$$\Delta f B_2 = P_{12}$$

Solving for Δf we get

$$f = \frac{-\Delta P_{L1}}{S_1 + S_2}$$

$$P_{12} = \frac{-\Delta P_{L1} S_2}{S_1 + S_2}$$

Thus

an increase of load in area 1 reduces frequency in both areas. Similarly for a change in load P_{L2} in area 2,

$$f = \frac{-\Delta P_{L2}}{S_1 + S_2}$$

and $P_{12} = P_{21} = \frac{-\Delta P_{L2} S_1}{S_1 + S_2}$

Example 5

A two area system connected by a tie-line has following parameters on 1000 MVA base;

$$R_1 = 4.5\%; \quad D_1 = 0.6; \quad H_1 = 4.5;$$

$$R_2 = 6\%; \quad D_2 = 0.85; \quad H_2 = 5.0;$$

The units are running in parallel at a frequency of 50 Hz. The synchronizing power coefficient is 1.9 pu at the initial operating angle. A load change of 150 MW occurs in area 1. Determine the new steady state frequency and the change in tie-line power flow.

Solution:

$$P_{L1} = \frac{150}{1000} = 0.15 \text{ pu.}$$

Steady state frequency deviation is

$$f = \frac{-\Delta P_{L1}}{S_1 + S_2}$$

$$1 = \frac{1}{R_1} + D_1 = \frac{1}{0.045} + 0.6 = 22.822$$

$$2 = \frac{1}{R_2} + D_2 = \frac{1}{0.06} + 0.85 = 17.516$$

$$f = \frac{-0.15}{22.822 + 17.516} = -0.0037 \text{ pu}$$

$$\text{Steady state frequency} = 50 + (0.0037 \times 50) = 49.815 \text{ Hz.}$$

$$P_{12} = f_2 = 0.0037 \times 17.516 = 0.0648 \text{ pu} = 64.8 \text{ MW}.$$

Since P_{12} is negative, it implies that 64.8 MW flows from area 2 to area 1.

Change in mechanical powers is given by

$$P_{m1} = \frac{-\Delta f}{R_1} = \left(\frac{-0.0037}{0.045} \right) = 0.082 \text{ pu} = 82 \text{ MW}$$

$$P_{m2} = \frac{-\Delta f}{R_2} = \left(\frac{-0.0037}{0.06} \right) = 0.0617 \text{ pu} = 61.7 \text{ MW}$$

Change in load in area 1 due to frequency sensitive loads is $fD_1 = (0.0037)(0.6) = 0.0022 \text{ pu} = 2.2 \text{ MW}$. Similarly for area 2 $fD_2 = (0.0037)(0.85) = 0.0031 \text{ pu} = 3.1 \text{ MW}$. Total change in load is 5.3 MW. The power flow of 64.8 MW from area 2 to area 1 is contributed by an increase in generation of area 2 by 61.7 MW and reduction in load of area 2 by 3.1 MW.

Tie-line bias control:

From discussion it can be seen that, if the areas are equipped only with primary control of the ALFC, a change in load in one area met is with change in generation in both areas, change in tie-line power and a change in the frequency. Hence, a supplementary control is necessary to maintain

- Frequency at the nominal value
- Maintain net interchange power with other areas at the scheduled values
- Let each area absorb its own load

Hence, the supplementary control should act only for the areas where there is a change in load. To achieve this, the control signal should be made up of the tie-line flow deviation plus a signal proportional to the frequency deviation. A suitable proportional weight for the frequency deviation is the frequency – response characteristic. This is the reason why is also called the frequency bias factor. This control signal is called the area control error (ACE). In a two area system

$$ACE_1 = P_{12} + B_1 f; \quad B_1 = \frac{1}{R_1}$$

$$ACE_2 = P_{21} + B_2 f; \quad B_2 = \frac{1}{R_2}$$

The ACE represents the required change in area generation and its unit is MW. ACEs are used as control signals to activate changes in the reference set points. Under

steady state P_{12} and f will be zero. The block diagram with the supplementary control is shown below. It is applied to selected units in each area.

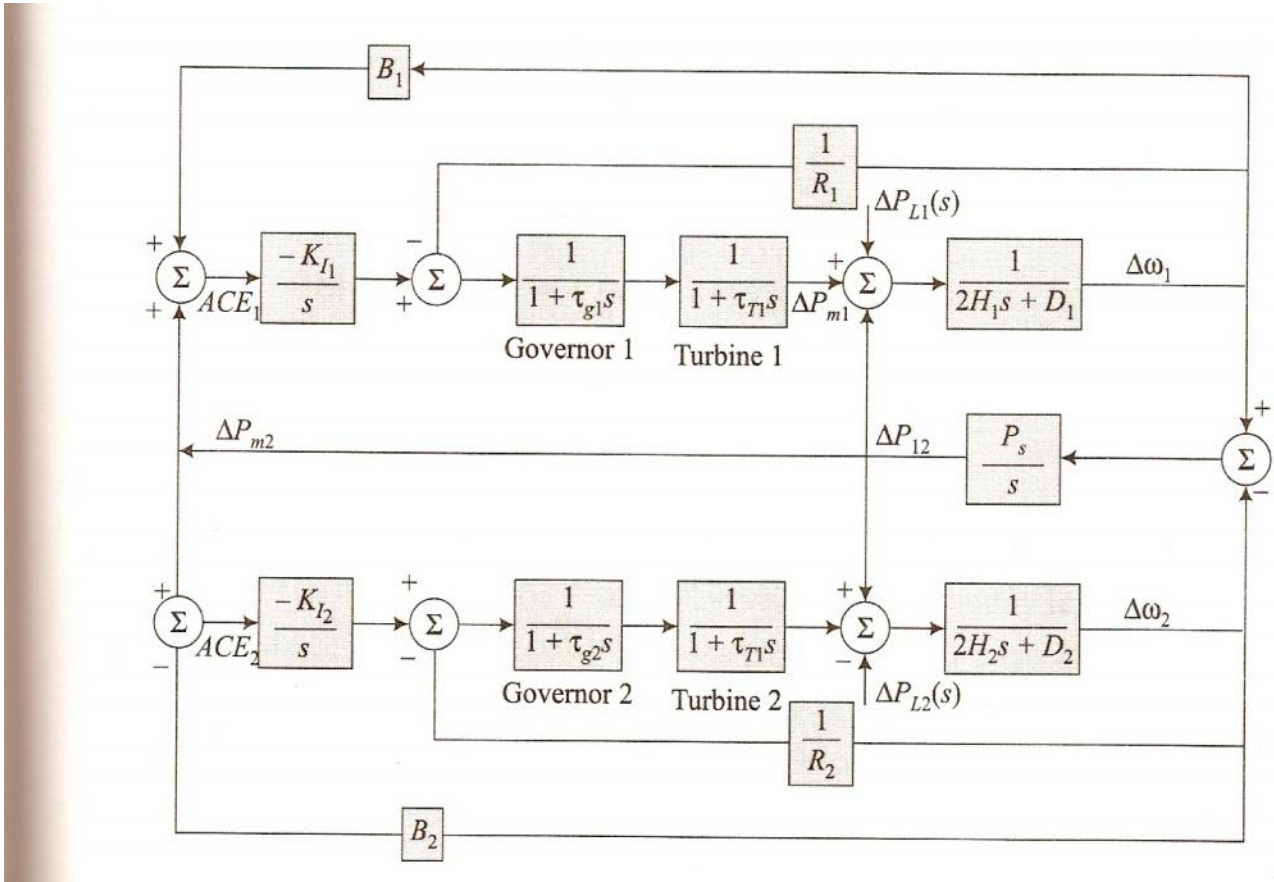


Fig : Block diagram with supplementary control

The operation of the ACE can be explained as follows. Consider an increase in load of area 1, which leads to a decrease in system frequency. The primary ALFC loop limits the frequency deviation to

$$f = \frac{-\Delta P_{L1}}{S_1 + S_2}$$

The tie-line power has a deviation $P_{12} = -2 f$. The slower acting supplementary control, starts responding now.

$$ACE_1 = P_{12} + B_1 f = \frac{(S_1 + S_2)}{(S_1 + S_2)}(-\Delta P_{L1}) = P_{L1}$$

$$\text{and } ACE_2 = P_{12} + B_2 f = \frac{(-\Delta P_{L1})}{(S_1 + S_2)}(-S_2 + S_2) = 0$$

Thus only supplementary control of area 1 responds to P_{L1} and the generation changed so that ACE_1 becomes zero.

Example 6:

Two areas are connected via an inter tie. The load at 50Hz, is 15,000 MW in area 1 and 35,000 MW in area 2. Area 1 is importing 1500 MW from area 2. The load damping constant in each area is $D = 1.0$ and the regulation $R = 6\%$ for all units. Area 1 has a spinning reserve of 800 MW spread over 4000 MW of generation capacity and area 2 has a spinning reserve of 1000 MW spread over 10,000 MW generation. Determine the steady state frequency, generation and load of each area and tie–line power for

- (a) Loss of 1000 MW in area 1, with no supplementary control
- (b) Loss of 1000 MW in area 1, with supplementary controls provided on generators with reserve.

$$B_1 = 250 \text{ MW/0.1 Hz and } B_2 = 400 \text{ MW/0.1 Hz}$$

Solution:

(a) Assume a lossless system

- Area 1: Load = 15,000 MW
 Power import from area 2 = 1,500 MW
 Generation = 15,000 – 1500 = 13,500 MW
 Reserve = 800 MW
 Total generation capacity = 13,500 + 800 = 14,300 MW
- Area 2: Load = 35,000 MW
 Export to area 1 = 1,500 MW
 Generation = 35,000 + 1,500 = 36,500 MW
 Reserve = 1000 MW
 Total generation capacity = 36,500 + 1000 = 37,500 MW

Regulation of 6% on a generation capacity of 14,300 MW (including reserve) corresponds to

$$R_1 = \frac{0.06 \times 50}{14300}$$

$$\frac{1}{R_1} = \frac{1}{0.06} \times \frac{14300}{50} = 4,766.67 \text{ MW/Hz}$$

Similarly

$$\frac{1}{R_2} = \frac{1}{0.06} \times \frac{37500}{50} = 12,500 \text{ MW/Hz}$$

$$\frac{1}{R_{eq}} = \frac{1}{R_1} + \frac{1}{R_2} = 17,266.6 \text{ MW/Hz.}$$

Load damping $D = 1.0$, which means a 1% change in load occurs for a 1% change in frequency. D_1 is computed for area 1 on a total load of 14000 MW, considering the loss of 1000 MW.

$$D_1 = 1 \times \frac{14000}{100} \times \frac{100}{50} = 280 \text{ MW/Hz.}$$

Similarly D_2 is calculated on a load of 35,000 MW since there is no change in load of area 2.

$$D_2 = 1 \times \frac{35000}{100} \times \frac{100}{50} = 700 \text{ MW/Hz.}$$

$$D_{eq} = D_1 + D_2 = 980 \text{ MW/Hz.}$$

The change in system frequency is given by

$$f = \frac{-\Delta P_L}{R_{eq} + D_{eq}} = \frac{-(-1000)}{17,266.6 + 980} = 0.0548 \text{ Hz.}$$

Load changes due to load damping are,

$$P_{D1} = D_1 \quad f = 280 \times 0.0548 = 15.344 \text{ MW}$$

$$P_{D2} = D_2 \quad f = 700 \times 0.0548 = 38.36 \text{ MW}$$

Change in generations are

$$P_{G1} = \frac{-\Delta f}{R_1} = 4766.6 \times 0.0548 = 261.2 \text{ MW}$$

$$P_{G2} = \frac{-\Delta f}{R_2} = 12500 \times 0.0548 = 685 \text{ MW}$$

We recalculate powers as follows

Area 1: New load = 15,000 1,000.00 + 15.344 = 14,015.344 MW

 New generation = 13,500 – 261.2 = 13,238.8 MW

 Deficit = 14,015.34 13,238.8 = 776 MW

Area 2: New load = 35,000 + 38.36 = 35,038.36 MW

 New generation = 36,500 – 685 = 35,815 MW

 Excess = 35,815 35,038.36 = 776 MW

Thus tie–line power is 776 MW and flows from area 2 to area 1.

Steady state frequency = 50 + 0.0548 = 50.0548 Hz

(b) With supplementary control and $R_1 = 250 \text{ MW}/0.1 \text{ Hz}$ and $R_2 = 400 \text{ MW}/0.1 \text{ Hz}$

Generating capacity with supplementary control in area 1 is 4000 MW (on reserve) and in area 2 it is 1000 MW. These supplementary controls will keep ACE_1 and ACE_2 at zero.

$$ACE_1 = R_1 \Delta f + P_{12} = 0$$

$$ACE_2 = R_2 \Delta f + P_{21} = R_2 \Delta f - \Delta P_{12} = 0$$

This means $P_{12} = 0$ and $\Delta f = 0$

Thus the load and generation in area 1 are reduced by 1000 MW. There is no steady state deviation of tie–line power flow and frequency. The generation and load of area 2 also do not change.